

Theoretical Study on Rotation of Doubly Layered Torque Balance Cable

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Abstract - Remotely Operated Vehicle (ROV) uses long tether cable for power supply and signal communication between a ROV and its support vessel. However there have been many accidents on this tether cable, such as kink and disconnection caused by rotation of the cable.

The tether cable is usually a torque balance type one with its strength members of inner/outer layer braided reverse direction each other. However, even if a torque balance of the cable is perfect, cable rotation occurs in most ROVs while paying in/out of a tether cable into the ocean.

In this research, calculation method of strand track of a tether cable bent on a sheave was established, and the reason of the cable rotation was found to be the change of track length caused by bending of the tether cable on sheave, applying this calculation method. Comparison of the calculated results and measured data of JAMSTEC ROVs shows this calculation method to be very good to explain the rotation phenomena.

I. INTRODUCTION

Although a very long tether cable is widely used for an operation of ROV for underwater development, many troubles on cable have been occurring in most ROVs in the world such as kink, birdcaging or disconnection. The Japan Marine Science and Technology Center (JAMSTEC) had faced this problem since the start of the operation of 3,300m class ROV “Dolphin 3K” (1985). Although JAMSTEC is now operating 11,000m class ROV “KAIKO,” this cable rotation problem had never been clarified.

In the development of the “KAIKO,” it was presumed that the fleet angle had influenced cable rotation phenomenon. Then a ram-type heave compensator with almost 0 degree of fleet angle was developed. However, the rotation of tether cable was still about 1 time per 100m, and the effect of the effort was not found [1, 2]. Although the number of cable rotation of the newer KAIKO No.2 cable decreased to 1/4 of the former one, the mechanism of this phenomenon has never been understood. Therefore the same problem must be inevitable when fabricating a long new cable.

Looking these phenomena, the authors obtained a new hypothesis that the reason of cable rotation must be a change of strand track while the cable is bent on sheaves from the straight condition. This hypothesis can only be proven by obtaining theoretically precise formulation to describe the strand tracks, and the authors had made the effort.

A. Cable Rotation Phenomena

Cable rotation described in this paper does not mean the rotation by residual torsion just after the fabrication. Fig.1 and Fig.2 are the examples of measured data of cable rotation to be discussed here. A weight with a Flux-gate compass is hunged at the bottom end of a cable, and the rotation number is counted while paying in/out of the cable into the sea. The rotation per unit cable length did not change by descending/ascending speed. The interesting point is the rotation of the weight while paying in of a tether cable. Any residual torsion, the cable may have, must be purged out when the cable is paying in, and the torque must be balanced all over the cable. Yet, the cable rotates by constant numbers per length while paying in from the sea. In addition to the number of rotation, the direction reverses while

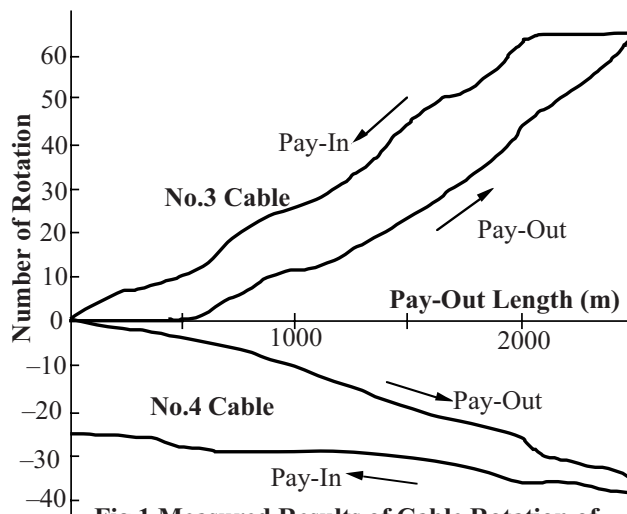


Fig.1 Measured Results of Cable Rotation of ROV Dolphin 3K No.3 and No.4 Cables

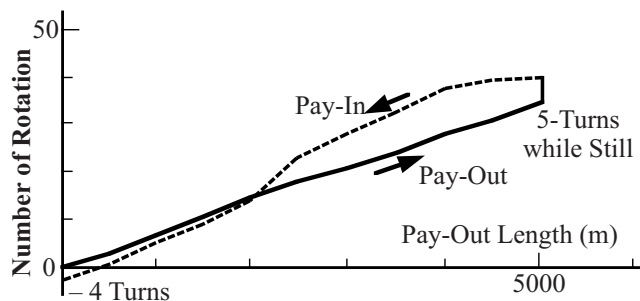


Fig.2 Measured Result of Cable Rotation of ROV KAIKO No.1 Cable

paying in and out.

This is a very strange phenomenon because there seems no unbalance torque to rotate the cable at all. There had been no clue to tackle and solve this problem, and only trial and error by fabricating and testing new cables had been continued.

B. Slip Restriction of Strands on Cable Rotation

Cable rotation seemed to be caused not either by cable or sheave only, but to be related to the interaction of cable and sheave. Especially, the outer strands and inner strands seemed to be slipping each other on the sheave. If they are really slipping, the rotation must occur. So, an adhesive stripe was introduced in between outer and inner strands of KAIKO No.1 cable in order to restrict the slip if any as shown in Fig.3. The test was done using a machine shown in Fig.4, where rotations of indication bars were measured. Comparison of the test results of a conventional cable and a cable with slip restriction are shown in Fig.5, and it shows the effect of slip restriction very clearly. The rotation of a conventional cable is about 1 rotation per 100m ($15\text{deg}/4\text{m} = 360\text{deg}/100\text{m}$) which is almost the same as shown in Fig.2, whereas, the cable with slip restriction would

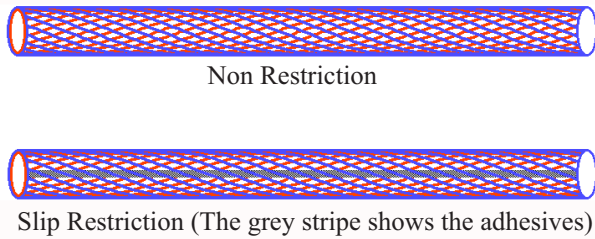


Fig.3 Slip Restriction on Strands

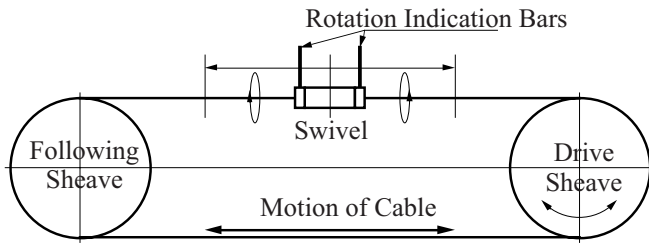


Fig.4 In-House Cable Rotation Measurement System

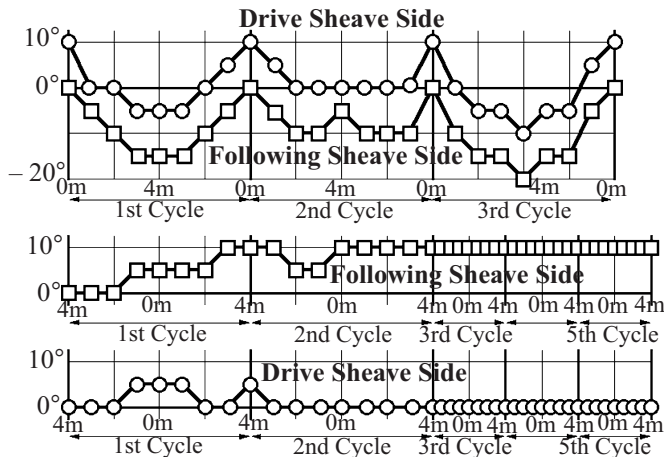


Fig.5 Results and Comparison of Slip Restriction Effect

not rotate.

This test gave the authors a clear idea on the cable rotation mechanism and the authors carried out further research for the purpose of clarifying this mechanism theoretically.

II. FUNDAMENTAL MECHANISM

Strands of doubly layered torque balance cable are braided reverse direction each other so that the torque of cable including power lines and signal lines is balanced. These strands in each layer are not contact one another, but are twisted around the cable core with adequate gap for flexibility of bending and twisting of the cable.

The tracks of these strands are helicoids with constant gradient in the straight state. However, when the cable is bent on a sheave, the each strand tends to go to inner part of bending to fill up the gap, and the gap at the outer part of bending becomes larger. The strand which contacts with the neighboring strands at the inner part of bending is called as "Contact strand," and the non-contact strand is called as "Free strand" in this paper. There must be a specific point on the strand where the contact strand detaches with the neighboring strands to become free strand. This point is called "Detaching point."

The track of the strand thus become different from that of the straight state, and the length of the strand at one pitch length must become different from that of the straight state also.

These differences must occur at both the inner and outer strands. The authors think that these differences destroy the torque balance of the cable and make the cable rotate.

So, the authors have started the effort to establish the calculation method of strands track of a tether cable bent on sheaves at first, and then to clarify the mechanism of cable rotation phenomena.

III. NOMENCLATURE

- R : Radius of sheave pitch circle
- p : Strand pitch
- D : Outer diameter of strand
- r : Radius of helical coil from centerline
- N : Number of strands
- $\mathbf{b}$: Binormal vector of strands and the components
- $\mathbf{N}$: Normal vector on curved surface of strands helicoidal coil and the components
- $\mathbf{n}$: Normal vector of strands track
- $\mathbf{t}$: Tangential vector of strands track and the components
- g : Cable rotation
- ν : Angle from binormal vector to cable center
- L : Strand/cable length ratio
- w : Radius of curvature of strands ($1/w$: curvature)
- q : Coordinate angle of helical coil for strand
- f : Phase angle from sheave center
- $*$: Vector product operator
- Δ : Difference between L_s
- subscript "i": Inner layer
- subscript "o": Outer layer
- subscript "s": Cable of straight condition

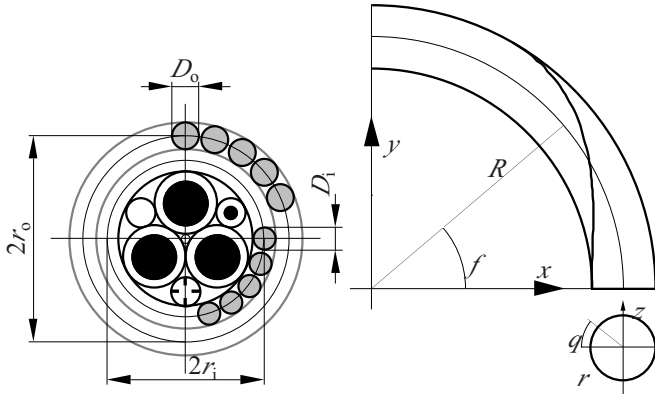


Fig.6 Cross Section of Cable and Symbols

Fig.7 Coordinate System of the Strand

subscript “c”: Cable of curved condition by bending on a sheave

Fig.6 shows the cross section of cable and symbols, and Fig.7 shows the coordinate system.

IV. TRACK CALCULATION[3, 4, 5]

In the coordinate system shown in Fig.7, coordinates of a strand center are express as follow:

$$(x, y, z) = \{(R - r \cos q) \cos f, (R - r \cos q) \sin f, r \sin q\} \quad (4.1)$$

By use of the infinitesimal increment of strand arc length $ds = (dx^2 + dy^2 + dz^2)^{1/2}$, the components of the tangential vector $t = (x_t, y_t, z_t) = (dx/ds, dy/ds, dz/ds)$ are given as follows:

$$\begin{aligned} dx/ds &= \{r \sin q \cos f - (R - r \cos q) \sin f (df/dq)\} / A \\ dy/ds &= \{r \sin q \sin f + (R - r \cos q) \cos f (df/dq)\} / A \\ dz/ds &= r \cos q / A \end{aligned} \quad (4.2)$$

where $A = \{r^2 + (R - r \cos q)^2 (df/dq)^2\}^{1/2}$.

Their second order differentials d^2x/ds^2 , d^2y/ds^2 , d^2z/ds^2 , are given as follow:

$$\begin{aligned} d^2x/ds^2 &= \{r^3 \cos q \cos f - 2r^3 \sin q \sin f (df/dq) \\ &\quad + r \cos f (R - r \cos q) (R \cos q - 2r) (df/dq)^2 \\ &\quad - r \sin q \sin f (R - r \cos q)^2 (df/dq)^3 - \cos f (R - r \cos q)^3 (df/dq)^4 \\ &\quad - r^2 \sin f (R - r \cos q) (d^2f/dq^2) \\ &\quad - \sin q \cos f (R - r \cos q)^3 (df/dq)^4\} / B \\ d^2y/ds^2 &= \{r^3 \cos q \sin f + 2r^3 \sin q \cos f (df/dq) \\ &\quad + r \sin f (R - r \cos q) (R \cos q - 2r) (df/dq)^2 \\ &\quad + r \sin q \cos f (R - r \cos q)^2 (df/dq)^3 - \sin f (R - r \cos q)^3 (df/dq)^4 \\ &\quad + r^2 \cos f (R - r \cos q) (d^2f/dq^2) \\ &\quad - \sin q \sin f (R - r \cos q)^3 (df/dq)^4\} / B \\ d^2z/ds^2 &= \{-r^3 \sin q - r \cos q (R - r \cos q)^2 (df/dq) (d^2f/dq^2) \\ &\quad - r R \sin q (R - r \cos q) (df/dq)^2\} / B \end{aligned} \quad (4.3)$$

where $B = \{r^2 + (R - r \cos q)^2 (df/dq)^2\}^{1/2}$.

Curvature $1/w$ of this strand track becomes as follows:

$$1/w = \{(d^2x/ds^2)^2 + (d^2y/ds^2)^2 + (d^2z/ds^2)^2\}^{1/2}.$$

The normal vector of this surface of the envelope of track is a unit vector shown as follows:

$$N = (x_N, y_N, z_N) = (\cos q \cos f, \cos q \sin f, -\sin q)$$

A. Free Strand

Free strand received tension only and does not receive external forces from the neighboring strands. So, the direction of normal vector of strand $n = (d^2x/ds^2, d^2y/ds^2, d^2z/ds^2)$ and that

of N are the same. Thus, the scalar product nnN is as follows:

$$\begin{aligned} n \{ r \cos q \cos f / d^2x/ds^2 + r \cos q \sin f / d^2y/ds^2 \\ - r \sin q / d^2z/ds^2 \} = 1 \end{aligned} \quad (4.4)$$

This equation becomes the following one, and is modified to eq. (4.5):

$$\begin{aligned} \{r^2 + (R - r \cos q)^2 (df/dq)^2\} [r(R - r \cos q) (d^2f/dq^2) \\ + \sin q \{2r^2 + (R - r \cos q)^2 (df/dq)^2\} (df/dq)]^2 = 0 \\ r(R - r \cos q) (d^2f/dq^2) \\ + \sin q \{2r^2 + (R - r \cos q)^2 (df/dq)^2\} (df/dq) = 0 \end{aligned} \quad (4.5)$$

This second order differential equation can be transformed to the following first order differential equation.

$$df/dq = \pm r / (R - r \cos q) \{K^2 (R - r \cos q)^2 - 1\}^{1/2}. \quad (4.6)$$

where K is an integration constant given by any boundary condition. This equation can be integrated easily, and a free strand track is determined.

B. Contact Strand

If a strand contacts with neighboring strands, the value of scalar product nnN is not 1, and eq. (4.6) cannot be applied. Instead, the contact point of the strand is calculated by the following procedure:

The tangential and normal vectors at any point P_I of a strand are easily calculated using its coordinate (q, f) . The binormal vector b_I at the point can also be calculated as follow by using the N_I and t_I .

$$\begin{aligned} b_I &= N_I * t_I = (x_{bI}, y_{bI}, z_{bI}) \\ x_{bI} &= \{r \sin f + (R - r \cos q) \sin q \cos f (df/dq)\} / C \\ y_{bI} &= \{-r \cos f + (R - r \cos q) \sin q \sin f (df/dq)\} / C \\ z_{bI} &= (R - r \cos q) \cos q (df/dq) / C \end{aligned} \quad (4.7)$$

where $C = \{r^2 + (R - r \cos q)^2 (df/dq)^2\}^{1/2}$.

The contact point of neighboring strand $P_{II}(x_{II}, y_{II}, z_{II})$ is on the $N_I - b_I$ plane, and is at a distance (r) (Radius of helical coil from centerline) apart from the cable center and at a distance (D) from P_I . The contact point $P_{II}(x_{II}, y_{II}, z_{II})$ is calculated as follows:

$$(x_{II}, y_{II}, z_{II}) = (x_I, y_I, z_I) + D \cos \nu_I b_I + D \sin \nu_I N_I. \quad (4.8)$$

where the ν_I means an angle from the strand center P_I on the $N_I - b_I$ plane to make the distance of P_{II} to the cable center as r as shown in Fig.8.

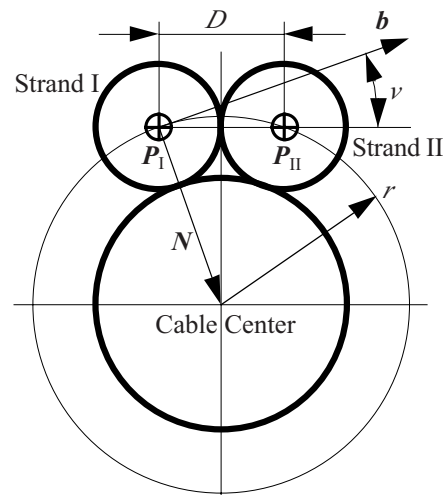


Fig.8 Calculation Method of Contact Point

section by section until the $f_{\pi} > p/2R$

- (2) The detaching point must exist on the middle of the latest section, and the point is surveyed so that the $f_{\pi} = p/2R$. Thus, the track of the strand is finally fixed.
- (3) These calculations are done both for inner layer strand and outer layer strand. The lengths of strands of both layers are also calculated

V. NUMBER OF CABLE ROTATION[5]

The strand lengths of the bent cable on a sheave are different from those of straight cable, and this difference must be the origin of the cable rotation. In order to evaluate the number of cable rotation, a new index L of the ratio of strand length to cable length is introduced. For the cable of straight condition, it is written as follows:

$$L_s = (p^2 + 4\pi^2 r^2)^{1/2}/p. \quad (5.1)$$

In the case of the cable bent on a sheave, the new index L = L_c is given as:

$$L_c = l_c/p. \quad (5.2)$$

where l_c is a strand length of one pitch length of cable on a sheave.

These indices mean strand lengths in unit lengths of cable.

When the cable in the straight condition proceeds into a sheave every unit length, the strand enters into the sheave by the length of L_s . However, the length of strand on sheave per unit length of cable is L_c . Most cases, $L_s \neq L_c$ and unbalance of strand length occurs by $\Delta L = L_s - L_c$.

For the cable of doubly layered strands, the unbalance of inner layer is $\Delta L_i = L_{si} - L_{ci}$, and that of outer layer is $\Delta L_o = L_{so} - L_{co}$.

If the differences of ΔL_i and ΔL_o are the same, inner and outer layer may only extend or shrink at the same rate. However, if the ΔL_i is different from ΔL_o , the unbalance of the length of an inner and outer layer occurs, and this unbalance must be

absorbed by some way, that means, the cable rotation.

The cable cannot rotate on a sheave since it is strongly pushed to sheaves by its tension, so the unbalance can be absorbed in the straight part because there are not any restrictions.

The rotation of g per unit length makes the pitch of both outer and inner layers as follow:

$$\begin{aligned} 1/p_o' &= 1/p_o + g \\ 1/p_i' &= 1/p_i - g. \end{aligned} \quad (5.3)$$

The length difference of the strands between inner and outer layers per unit length of the cable by this rotation g can be calculated as follows:

$L_{si}' = (p_i'^2 + 4\pi^2 r_i^2)^{1/2}/p_i'$: The L_s of inner strand after rotation of g

$L_{so}' = (p_o'^2 + 4\pi^2 r_o^2)^{1/2}/p_o'$: The L_s of outer strand after rotation of g

By the cable rotation of g , the L_s of the inner and outer strand change by ΔL_{si} and ΔL_{so} as follow:

$$\begin{aligned} \Delta L_{si} &= L_{si}' - L_{si} = (L_{si}'^2 - L_{si}^2)/(L_{si}' + L_{si}) \\ &\approx (L_{si}'^2 - L_{si}^2)/2L_{si} \approx -4\pi^2 r_i^2 g/p_i L_{si} \\ \Delta L_{so} &= L_{so}' - L_{so} = (L_{so}'^2 - L_{so}^2)/(L_{so}' + L_{so}) \\ &\approx (L_{so}'^2 - L_{so}^2)/2L_{so} \approx +4\pi^2 r_o^2 g/p_o L_{so} \end{aligned}$$

In order for the cable to absorb the length difference, the difference $\Delta L_{so} - \Delta L_{si}$ at the cable of straight condition must be equal to that of the bent condition on a sheave, that means, $\Delta L_o - \Delta L_i$. This gives the following equation:

$$\begin{aligned} \Delta L_o - \Delta L_i &= \Delta L_{so} - \Delta L_{si} = 4\pi^2 (r_i^2/p_i L_{si} + r_o^2/p_o L_{so})g. \\ g &= (\Delta L_o - \Delta L_i)/4\pi^2 (r_i^2/p_i L_{si} + r_o^2/p_o L_{so}) \end{aligned} \quad (5.4)$$

The term of $\Delta L_o - \Delta L_i$ is calculated with $g = 0$ at the start, and a repetition calculation is required so that the same g is used both on $\Delta L_{so} - \Delta L_{si}$ and on $\Delta L_o - \Delta L_i$. Further, the actual rotation becomes $2g$ because the cable of rotation g enters into the sheave with further rotation of g at the entrance to the sheave. Or, while getting off the sheave, the cable releases the rotation g at the exit of the sheave and releases further rotation of g of its own twisting.

VI. JAMSTEC TETHER CABLE DATA AND THE CALCULATION RESULTS

JAMSTEC has been and is now using several long tether cables and their specifications are shown in Table 1. The comparison between the measured rotation data of these cables in the sea and calculated results of them are shown in the same Table 1. The calculated results agree well with measured data except those of the Dolphin 3K-No.4 cable and the KAIKO-No.1 cable (the colored columns). For the cases of good agreement, the strands are steel wire or Kevlar FRP both with exactly round sections as design data. This Kevlar cable has been fabricated by winding the solidified Kevlar strand around the cable core. On the other hand, for the cases of no good agreement, sections are not round shaped, but are like quadrilateral sections as shown in Fig.11. This cable has been fabricated by winding non-solidified Kevlar strand around the cable core, and then it was solidified. The data used for these cases were those before winding around the cable core.

So the cross-sectional shapes were measured using actual

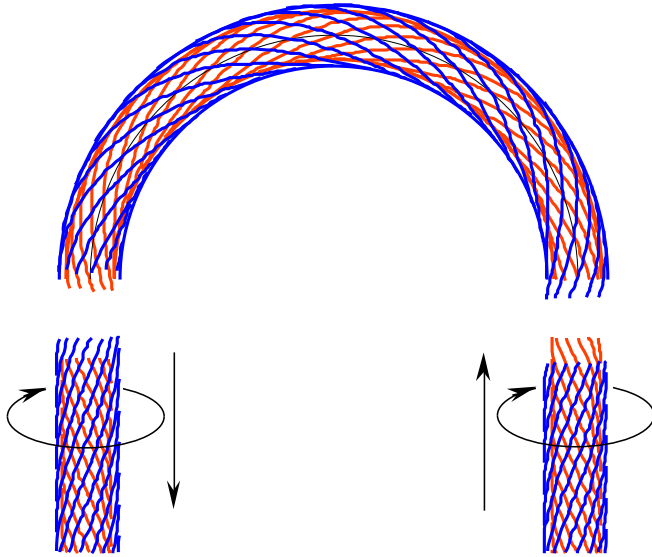


Fig.10 Cable Rotation at the Entrance and Exit of Sheave

The Difference of Inner and Outer Layers makes the Cable outside of Sheave rotate.



Fig.11 Cross-Section of KAIKO-No.1 Cable

Strands are quadrilateral shapes due to the close contact with each other while they are solidified.

cable samples. But it was nearly impossible to determine the contact point with neighboring strands. Calculations with various contact points were done, and it was found that calculation results agreed with measured data of cable rotation by selecting the contact point appropriately as shown in Table 2. It must be noted that the Dolphin 3K-No.4 cable rotates reverse direction compared with the other cables and this reverse direction of rotation could happen when the contact point or the gap is an appropriate value.

VII. CONCLUSION

The cable rotation phenomena of doubly layered torque balance cable can be explained by the previous description. The cause of cable rotation is summarized as follow:

- (1) Each strand of a tether cable has adequate gap between the neighboring strands in order to keep its flexibility.
- (2) If a cable is bent on a sheave, each strand tends to go to the inner side of bending to fill-up these gaps under tension. This makes the strand as "Contact Strand."
- (3) The contact state of a strand with the neighboring strands

Table 1. Specifications and Measured/Calculated Rotation of JAMSTECs ROV Cables

		D-3K No.3	D-3K No.4	D-3K No.5	KAIKO No.1	KAIKO No.2	Deep Tow No.1	Deep Tow No.2	Hyper-D
Cable Dia.(mm)		30.5	30.3	30.4	44.0	43.7	17.4	17.4	27.7
Strand Pitch(mm)	Outer L.	400	300	282	400	480	125	118	194
	Inner L.	250	245	250	250	380	99	100	173
Strand Dia.(mm)	Outer L.	1.9	2.0	2.2	3.6	3.05	1.84	1.43	1.47
	Inner L.	1.9	2.5	2.5	4.3	4.55	1.46	1.59	2.01
Winding Pitch Rad.(mm)	Outer L.	12.8	12.85	12.9	18.7	18.6	7.78	7.985	12.99
	Inner L.	10.9	10.6	10.55	14.75	14.8	6.13	6.475	11.25
Number of Strands	Outer L.	39	35	30	30	36	24	31	50
	Inner L.	33	24	22	20	19	24	23	32
Pitch Circle Rad. of Sheave(mm)			462			700		477.5	610.8
Measured Rotation/100m		ab.+3	ab.-1	—	ab.+1	ab.+1/4	ab.+0.9	ab.+0.9	—
Calculated Rotation/100m		2.79	4.28	2.81	2.27	0.22	0.79	0.88	0.46

The Name "D-3K" is "Dolphin 3K" and "Hyper D" is "Hyper Dolphin."

ends at the certain point as it goes outer side of bending. Then the track changes to "Free Strand."

- (4) The tracks and lengths of the strands change due to the bending, and the lengths of inner and outer strands every unit length of cable become different from those at straight condition of cable.
- (5) These differences can only be absorbed by the cable rotation in the straight part.
- (6) The cable rotation can be expressed by the eq. (5.4), and the amount is $2g$.

Operation of long tether cable is inevitable for the ocean activities, and the cable rotation must be as small as possible. Torque balance is one of the most important items for the design and fabrication of the cable, but also great attention must be paid to the cable rotation on sheave, utilizing the concept shown in this paper.

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Table 2. Recalculation Results of Large Discrepancy Cables

		D-3K No.4	D-3K No.5	KAIKO No.1
Strand Thickness(mm)	Outer L.	1.88	1.76	3.5
	Inner L.	2.28	2.08	4.0
Strand Width(mm)	Outer L.	2.10	2.43	3.5
	Inner L.	2.56	2.92	4.1
Contacting Pitch Rad.(mm)	Outer L.	12.48	12.5	18.35
	Inner L.	10.7	10.9	14.7
Measured Rotation/100m		ab.-1	—	ab.+1
Calculated Rotation/100m		-1.03	+1.47	+1.02