



Press Fit Engineering Design Equations and Calculator

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Requires a Java Enabled Browser

(1) Inner Body Poisson Radial Displacement

$$u_{\text{Poisson}} = \frac{2F\eta_{\text{outer}} D_{\text{I,outer}}}{\pi(D_{\text{O,outer}}^2 - D_{\text{I,outer}}^2)E_{\text{outer}}} - \frac{2F\eta_{\text{inner}} D_{\text{O,inner}}}{\pi(D_{\text{O,inner}}^2 - D_{\text{I,inner}}^2)E_{\text{inner}}}$$

(2) Thermal Radial Mismatch

$$u_{\text{thermal}} = \Delta T (CTE_{\text{inner}} - CTE_{\text{outer}}) \cdot D_{\text{interface}} / 2, \text{ use } D_{\text{interface}} = D_{\text{I,outer}}$$

(3) Outer Body Radial Displacement Caused by Rotation

$$u_{\text{centrif,outer}} = \frac{\rho \omega^2}{8E^*} \left\{ -\left(\frac{D_{\text{I,outer}}}{2}\right)^2 + (3 + \eta_{\text{outer}}) \left[\frac{(D_{\text{O,outer}}^2 + D_{\text{I,outer}}^2)}{4(1 + \eta_{\text{outer}})} \cdot \frac{D_{\text{I,outer}}}{2} + \frac{D_{\text{O,outer}}^2 D_{\text{I,outer}}}{8(1 - \eta_{\text{outer}})} \right] \right\}$$

(4) Inner Body Radial Displacement Caused by Rotation

$$u_{\text{centrif,inner}} = \frac{\rho \omega^2}{8E^*} \left\{ -\left(\frac{D_{\text{O,inner}}}{2}\right)^2 + (3 + \eta_{\text{inner}}) \left[\frac{(D_{\text{O,inner}}^2 + D_{\text{I,inner}}^2)}{4(1 + \eta_{\text{inner}})} \cdot \frac{D_{\text{O,inner}}}{2} + \frac{D_{\text{O,inner}} D_{\text{I,inner}}^2}{8(1 - \eta_{\text{inner}})} \right] \right\}$$

(5) Interface Pressure as a result of Diametrical Interference

$$P = \frac{\Delta}{\frac{D_{\text{I,outer}}}{E_{\text{outer}}} \left(\frac{D_{\text{O,outer}}^2 + D_{\text{I,outer}}^2}{D_{\text{O,outer}}^2 - D_{\text{I,outer}}^2} + \eta_{\text{outer}} \right) + \frac{D_{\text{O,inner}}}{E_{\text{inner}}} \left(\frac{D_{\text{O,inner}}^2 + D_{\text{I,inner}}^2}{D_{\text{O,inner}}^2 - D_{\text{I,inner}}^2} - \eta_{\text{inner}} \right)}$$

For the Outer Body Subjected to Internal Pressure, Axial Force, Torque, and Rotation.

(6) Radial Displacement of the inner Surface Caused by Internal Pressure.

$$u_{\text{inner}} = \frac{D_{\text{I,outer}} P}{2E_{\text{outer}}} \left(\frac{D_{\text{I,outer}}^2 + D_{\text{O,outer}}^2}{D_{\text{O,outer}}^2 - D_{\text{I,outer}}^2} + \eta_{\text{outer}} \right)$$

(7) Radial Stress Caused by internal Pressure

$$\sigma_{r,\text{pressure}} = \frac{D_{\text{I,outer}}^2 P}{D_{\text{O,outer}}^2 - D_{\text{I,outer}}^2} \left(1 - \frac{D_{\text{O,outer}}^2}{D_{\text{I,outer}}^2} \right)$$

(8) Circumferential Stress Caused by Axial Force

$$\sigma_{\theta,\text{pressure}} = \frac{D_{\text{I,outer}}^2 P}{D_{\text{O,outer}}^2 - D_{\text{I,outer}}^2} \left(1 + \frac{D_{\text{O,outer}}^2}{D_{\text{I,outer}}^2} \right)$$

(9) Axial Stress Caused by Torque

$$\sigma_z = \frac{4F}{\pi(D_{\text{O,outer}}^2 - D_{\text{I,outer}}^2)}$$

(10) Shear Stress Caused by Torque

$$\tau = \frac{16TD_{\text{I,outer}}}{\pi(D_{\text{O,outer}}^4 - D_{\text{I,outer}}^4)}$$

(11) Circumferential Centrifugal Stress at the Inner Surface

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$$\sigma_{\theta, \text{centrifugal}} = \frac{\rho \omega^2 (3 + \eta_{\text{outer}})}{8} \left(\frac{D_{I, \text{outer}}^2}{4} + \frac{D_{O, \text{outer}}^2}{2} - \frac{(1 + 3\eta_{\text{outer}}) D_{I, \text{outer}}^2}{(3 + \eta_{\text{outer}}) 4} \right)$$

(12) Von Mises Stress at the Inner Surface

$$\sigma_{\text{VonMises}} = \sqrt{\frac{(\sigma_r - \sigma_{\theta, \text{pressure}} - \sigma_{\theta, \text{centrifugal}})^2 + (\sigma_{\theta, \text{pressure}} + \sigma_{\theta, \text{centrifugal}} - \sigma_z)^2 + (\sigma_r - \sigma_z)^2}{2}} + 3\tau^2$$

For the Inner Body Subjected to External Pressure, Axial Force, torque and Rotation

(13) The Radial Stress Caused by Internal Pressure

$$\sigma_r = -\frac{D_{O, \text{inner}} P}{2E_{\text{outer}}} \left(\frac{D_{O, \text{inner}}^2 - D_{I, \text{inner}}^2}{D_{O, \text{inner}}^2 - D_{I, \text{inner}}^2} - \eta_{\text{inner}} \right)$$

(14) Radial Stress Caused by Internal Pressure

$$\sigma_r, \text{pressure} = \frac{D_{O, \text{inner}}^2 P}{D_{O, \text{inner}}^2 - D_{I, \text{inner}}^2} \left(1 - \frac{D_{I, \text{inner}}^2}{D_{O, \text{inner}}^2} \right)$$

(15) Circumferential Stress Caused by Internal Pressure

$$\sigma_{\theta, \text{pressure}} = \frac{D_{O, \text{inner}}^2 P}{D_{O, \text{inner}}^2 - D_{I, \text{inner}}^2} \left(1 + \frac{D_{I, \text{inner}}^2}{D_{O, \text{inner}}^2} \right)$$

(16) Axial Stress Caused by Axial Force

$$\sigma_z = \frac{AF}{\pi(D_{O, \text{inner}}^2 - D_{I, \text{inner}}^2)}$$

(17) Shear Stress Caused by Axial Force

$$\tau = \frac{16TD_{O, \text{inner}}}{\pi(D_{O, \text{inner}}^4 - D_{I, \text{inner}}^4)}$$

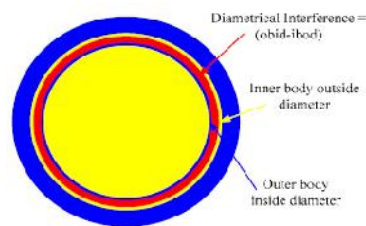
(18) Circumferential Centrifugal Stress at the outer Surface

$$\sigma_{\theta, \text{centrifugal}} = \frac{\rho \omega^2 (3 + \eta_{\text{inner}})}{8} \left(\frac{D_{I, \text{inner}}^2}{2} + \frac{D_{O, \text{inner}}^2}{4} - \frac{(1 + 3\eta_{\text{inner}}) D_{O, \text{inner}}^2}{(3 + \eta_{\text{inner}}) 4} \right)$$

(19) Von Mises Stress at the outer Surface

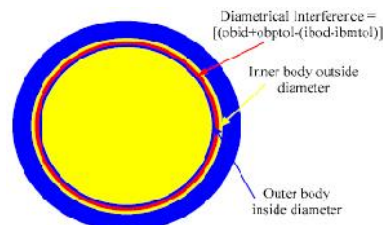
$$\sigma_{\text{VonMises}} = \sqrt{\frac{(\sigma_r - \sigma_{\theta, \text{pressure}} - \sigma_{\theta, \text{centrifugal}})^2 + (\sigma_{\theta, \text{pressure}} + \sigma_{\theta, \text{centrifugal}} - \sigma_z)^2 + (\sigma_r - \sigma_z)^2}{2}} + 3\tau^2$$

Ideal Case



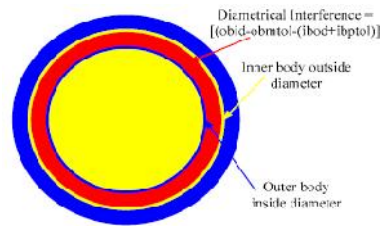
Worst Case for Loose Fit:

Joint may not be able to transmit desired force or torque



Worst Case for Tight Fit:

Yield stress may be exceeded and outer body may rupture



Reference:

Slocum, A.H. Precision Machine Design, 1995, Society of Manufacturing Engineers, Dearborn
Published by Prentice Hall, 1992. [Engineering Design Equations](#)



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